

**CANKAYA UNIVERSITY
FACULTY OF ENGINEERING
MECHANICAL ENGINEERING DEPARTMENT
ME 212 THERMODYNAMICS II**

CHAPTER 13

EXAMPLES SOLUTIONS

1) One hundred kmol of propane (C_3H_8) together with 3572 kmol of air enter a furnace per unit of time. Carbon dioxide, carbon monoxide, and unburned fuel appear in the products of combustion exiting the furnace. Determine the percent excess or deficiency of air, whichever is appropriate.



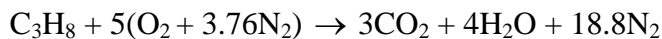
Assumptions:

- 1) 3.76 kmol of N_2 accompanying each kmol of O_2 in air.
- 2) N_2 is inert gas.

The data are provided for actual operation. Thus:

$$\overline{AF} = \frac{3572}{100} = 35.72 \frac{\text{kmol air}}{\text{kmol fuel}}$$

The balance equation for complete combustion with the theoretical amount of air is:



The theoretical air/fuel ratio is:

$$\left(\overline{AF}\right)_{\text{theo}} = \frac{5(4.76)}{1} = 23.8 \frac{\text{kmol air}}{\text{kmol fuel}}$$

Accordingly,

$$\% \text{ excess} = \left(\frac{35.72 - 23.8}{23.8} \right) (100) = 50\%$$

2) Propane (C₃H₈) is burned with air. For each case, obtain the balanced reaction equation for complete combustion.

a) with the theoretical amount of air.

b) with 20% excess air.

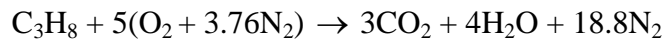
c) with 20% excess air, but only 90% of the propane being consumed in the reaction.

Assumptions:

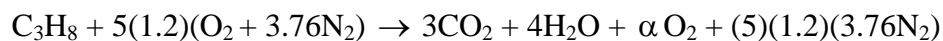
1) 3.76 kmol of N₂ accompanying each kmol of O₂ in air.

2) N₂ is inert gas.

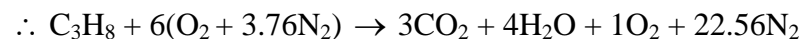
a) Complete combustion with the theoretical amount of air.



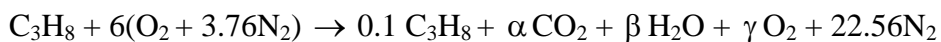
b) Complete combustion with 20% excess air:



$$\text{O: } (6)(2) = 6 + 4 + 2\alpha \Rightarrow \alpha = 1$$



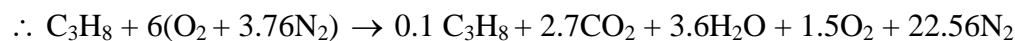
c) Combustion with 20% excess air. 90% of the fuel is burned.



$$\text{C: } 3 = 0.3 + \alpha \Rightarrow \alpha = 2.7$$

$$\text{H: } 8 = 0.8 + 2\beta \Rightarrow \beta = 3.6$$

$$\text{O: } 12 = 2(2.7) + 3.6 + 2\gamma \Rightarrow \gamma = 1.5$$



3) A fuel mixture with the molar analysis 70% CH₄, 20% CO, 5% O₂, and 5% N₂ burns completely with 20% excess air. Determine:

a) the balanced reaction equation.

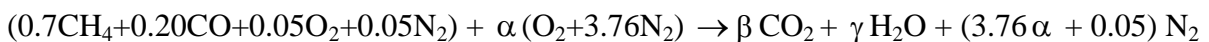
b) the air-fuel ratio, both on a molar and mass basis.

Assumptions:

1) 3.76 kmol of N₂ accompanying each kmol of O₂ in air.

2) N₂ is inert gas.

On the basis of 1 mol of fuel mixture, the reaction equation for complete combustion with the theoretical amount of air is:

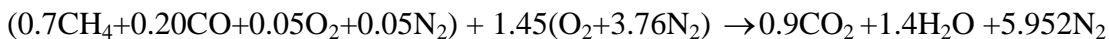


$$\text{C} : 0.7 + 0.2 = \beta \Rightarrow \beta = 0.9$$

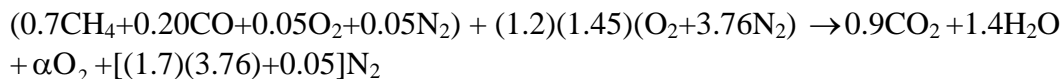
$$\text{H} : 2.8 = 2\gamma \Rightarrow \gamma = 1.4$$

$$\text{O} : 0.2 + 0.1 + 2\alpha = 2(0.9) + 1.4 \Rightarrow \alpha = 1.45$$

Thus,

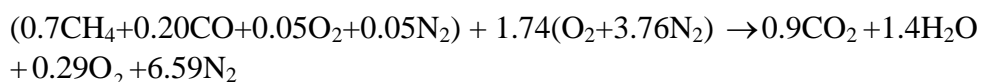


Thus the reaction for complete combustion with 20% excess air is:



$$\text{O} : 0.2 + 0.1 + 2(1.74) = 1.8 + 1.4 + 2\alpha \Rightarrow \alpha = 0.29$$

Accordingly;



b) The air fuel ratio is:

$$\overline{\text{AF}} = \frac{(1.74)(4.76)}{1} = 8.25 \frac{\text{kmol air}}{\text{kmol fuel}}$$

The fuel molecular mass is:

$$M_f = (0.7)(16.04) + (0.2)(28.01) + (0.05)(32) + 0.05(28.01) = 19.831$$

so;

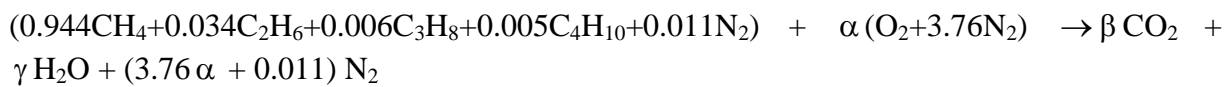
$$\text{AF} = \left(\overline{\text{AF}} \right) \left(\frac{M_{\text{air}}}{M_{\text{fuel}}} \right) = (8.25) \left(\frac{28.97}{19.831} \right) = 12.05 \frac{\text{kg air}}{\text{kg fuel}}$$

4) A fuel mixture with the molar analysis 94.4% CH₄, 3.4% C₂H₆, 0.6% C₃H₈, 0.5% C₄H₁₀, 1.1% N₂ burns completely with 20% excess air in a reactor operating at steady state. If the molar flow rate of the fuel is 0.1 kmol/h, determine the molar flow rate of the air, in kmol/h.

Assumptions:

- 1) 3.76 kmol of N₂ accompanying each kmol of O₂ in air.
- 2) N₂ is inert gas.

On the basis of 1 mol of fuel mixture, complete combustion with the theoretical amount of air is:



$$\text{C} : 0.944 + 2(0.034) + 3(0.0006) + 4(0.005) = \beta \Rightarrow \beta = 1.05$$

$$\text{H} : 4(0.944) + 6(0.034) + 8(0.0006) + 10(0.005) = 2\gamma \Rightarrow \gamma = 2.039$$

$$\text{O} : 2\alpha = 2(1.05) + 2.039 \Rightarrow \alpha = 2.0695$$

Combustion with 20% excess air means that the molar flow rate is:

$$\overline{\text{AF}} = \frac{1.2(2.0695)(4.76)}{1} = 11.82 \frac{\text{kmol air}}{\text{kmol fuel}}$$

Thus, with $\dot{n}_{\text{fuel}} = 0.1 \text{ kmol/h}$

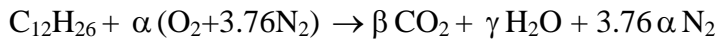
$$\dot{n}_{\text{air}} = 1.182 \text{ kmol air/h}$$

- 5) Dodecane ($C_{12}H_{26}$) burns completely with 150% of theoretical air. Determine:
- the air-fuel ratio on a molar and mass basis.
 - the dew point temperature of the combustion products, in $^{\circ}C$, when cooled at 1 atm.

Assumptions:

- 3.76 kmol of N_2 accompanying each kmol of O_2 in air.
- N_2 is inert gas.

- For complete combustion with the theoretical amount of air:

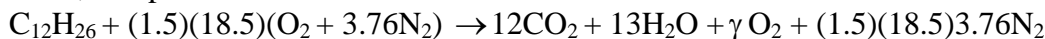


$$C : 12 = \beta$$

$$O : 2\alpha = 2(12) + 13 \Rightarrow \alpha = 18.5$$

$$H : 26 = 2\gamma \Rightarrow \gamma = 13$$

Then, complete combustion with 150% of the theoretical amount of air:



$$O : 2(1.5)(18.5) = 24 + 13 + 2\gamma \Rightarrow \gamma = 9.25$$

Accordingly:

$$\overline{AF} = \frac{(1.5)(18.5)(4.76)}{1} = 132.09 \frac{\text{kmol air}}{\text{kmol fuel}}$$

$$AF = \left(\overline{AF}\right) \left(\frac{M_{\text{air}}}{M_{\text{fuel}}}\right) = (132.09) \left(\frac{28.97}{(12)(12.01) + (26)(1.009)}\right) = 22.46 \frac{\text{kg air}}{\text{kg fuel}}$$

- The partial pressure of the water in the combustion products is

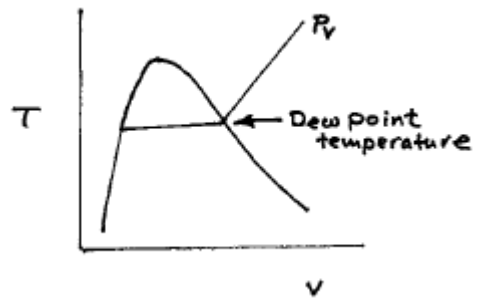
$$p_v = Y_v p_1 \text{ where;}$$

$$Y_v = \frac{13}{(12 + 13 + 9.25 + 104.34)} = 0.0938$$

so

$$p_v = (0.0938)(1.01325 \text{ bar}) = 0.09504 \text{ bar}$$

Then, from tables: $T_{\text{dew}} = 44.8^{\circ}C$



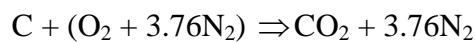
6) Carbon burns with 80% theoretical air yielding CO₂, CO, and N₂ only. Determine:

- the balanced reaction equation.
- the air-fuel ratio on a mass basis.
- the analysis of the products on a molar basis.

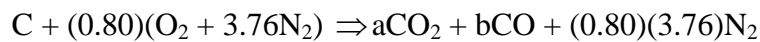
Assumptions:

- 3.76 moles of N₂ accompanying each mole of O₂ in air.
- N₂ is inert gas.

a) Complete combustion of C with the theoretical amount of air is described by:



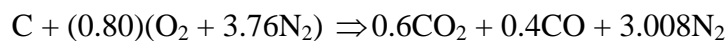
Thus, combustion with 80% theoretical air produces CO₂, CO, and N₂ is:



$$C : 1 = a + b \Rightarrow b = 1 - a$$

$$O : (0.80)(2) = 2a + b \Rightarrow 1.6 = 2a + (1 - a) \Rightarrow a = 0.6 \Rightarrow b = 0.4$$

Accordingly, the balanced reaction equation is:



b) The air fuel ratio is:

$$\overline{AF} = \frac{(0.80)(4.76)}{1} = 3.808 \frac{\text{kmol air}}{\text{kmol fuel}}$$

$$AF = (\overline{AF}) \left(\frac{M_{\text{air}}}{M_{\text{fuel}}} \right) = (3.808) \left(\frac{28.97}{12.01} \right) = 9.19 \frac{\text{kg air}}{\text{kg fuel}}$$

c) The molar analysis of the products is:

$$\%CO_2 = \left(\frac{0.6}{4.008} \right) (100) = 15\%$$

$$\%CO = \left(\frac{0.4}{4.008} \right) (100) = 10\%$$

$$\%N_2 = \left(\frac{3.008}{4.008} \right) (100) = 75\%$$

7) Liquid methanol (CH_3OH) burns with air. The product gas is analyzed and the laboratory report gives only the following percentages on a dry molar basis: 7.1% CO_2 , 2.4% CO , and 0.84% CH_3OH . Assuming the balance consists of O_2 and N_2 , determine:

- a) the percentage of O_2 and N_2 in the dry molar analysis.
 b) the percent excess air.

Assumptions:

- 1) 3.76 moles of N_2 accompanying each mole of O_2 in the combustion air.
 2) N_2 is inert gas.
 3) The dry products include only CO_2 , CO , CH_3OH , O_2 and N_2 .

a) On the basis of 100 moles of dry products:



Note : For the dry products, $7.1 + 2.4 + 0.84 + x + z = 100 \Rightarrow x + z = 89.66$, or $z = 89.66 - x$

$$\text{C} : y = 7.1 + 2.4 + 0.84 \Rightarrow y = 10.34$$

$$\text{H} : 4(10.34) = (0.84)(4) + 2w \Rightarrow w = 19$$

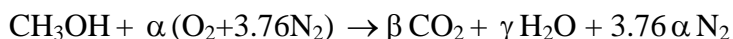
$$\text{O} : (10.34) + 2\alpha = (7.1)(2) + 2.4 + 0.84 + 2(89.66 - x) + 19 \Rightarrow \alpha = 102.71 - x$$

$$\text{N} : \left. \begin{array}{l} (\alpha)(2)(3.76) = 2x \\ \Rightarrow 3.76\alpha = x \end{array} \right\} \alpha = 102.71 - 3.76\alpha \Rightarrow \alpha = 21.578 \Rightarrow x = 81.13 \text{ and } 89.66 - x = 8.53$$

Accordingly;

$$\% \text{N}_2 = 81.13 \text{ and } \% \text{O}_2 = 8.53$$

b) For complete combustion with the theoretical amount of air



$$\text{C} : 1 = \beta$$

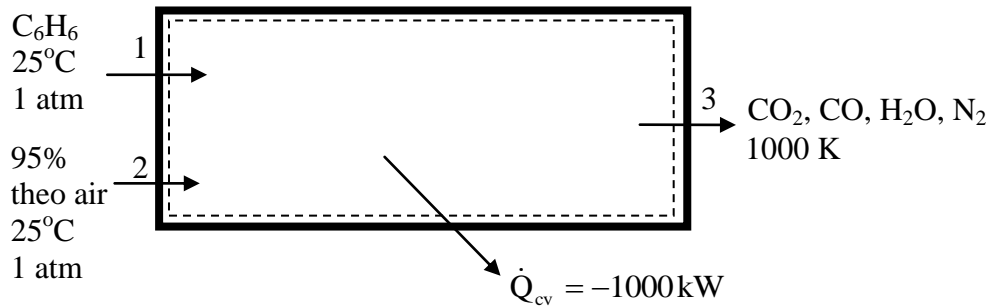
$$\text{H} : 4 = 2\gamma \Rightarrow \gamma = 2$$

$$\text{O} : 1 + 2\alpha = 2(1) + 2 \Rightarrow \alpha = 3/2$$

$$\left(\begin{array}{c} \% \text{ theoretical} \\ \text{air} \end{array} \right) = \left(\frac{(21.578)(4.76) / 10.34}{(1.5)(4.76) / 1} \right) (100) = 139$$

$$\Rightarrow \% \text{ excess air} = 39$$

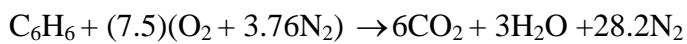
8) Benzene gas (C₆H₆) at 25°C, 1 atm enters a combustion chamber operating at steady state and burns with 95% theoretical air entering at 25°C, 1 atm. The combustion products exit at 1000 K and include only CO₂, CO, H₂O, and N₂. Determine the mass flow rate of the fuel, in kg/s, to provide heat transfer at a rate of 1000 kW.



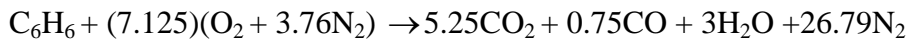
Assumptions:

- 1) The control volume shown is at steady state with $\dot{W}_{cv} = 0$ and negligible kinetic/potential energy effects.
- 2) 3.76 moles of N₂ accompany each mole of O₂ in the combustion air and N₂ is inert.
- 3) The ideal gas model is applicable to the combustion air and the products of combustion.

The balanced reaction equation for combustion with the theoretical amount of air is:



The balanced reaction equation for combustion with 95% theoretical air is:



An energy rate balance reduces to:

$$0 = \frac{\dot{Q}_{cv}}{\dot{n}_{\text{fuel}}} - \underbrace{\frac{\dot{W}_{cv}}{\dot{n}_{\text{fuel}}}}_{=0} + (\bar{h}_{\text{fuel}})_1 + [7.125\bar{h}_{\text{O}_2} + 26.79\bar{h}_{\text{N}_2}]_2 - [5.25\bar{h}_{\text{CO}_2} + 0.75\bar{h}_{\text{CO}} + 3\bar{h}_{\text{H}_2\text{O}} + 26.79\bar{h}_{\text{N}_2}]_3$$

$$\text{with } \bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$$

$$\frac{\dot{Q}_{cv}}{\dot{n}_{\text{fuel}}} = 5.25[\bar{h}_f^\circ + \Delta\bar{h}]_{\text{CO}_2} + 0.75[\bar{h}_f^\circ + \Delta\bar{h}]_{\text{CO}} + 3[\bar{h}_f^\circ + \Delta\bar{h}]_{\text{H}_2\text{O}} + 26.79[\bar{h}_f^\circ + \Delta\bar{h}]_{\text{N}_2} - (\bar{h}_f^\circ)_{\text{fuel}}$$

with data from the ideal gas tables:

$$\frac{\dot{Q}_{cv}}{\dot{n}_{fuel}} = 5.25[-393520 + 42769 - 9364] + 0.75[-110530 + 30355 - 8669] \\ + 3[-241820 + 35882 - 9904] + 26.79[30129 - 8669] - 82930$$

$$\Rightarrow \frac{\dot{Q}_{cv}}{\dot{n}_{fuel}} = -2112780 \text{ kJ / kmol fuel}$$

Then, for $\dot{Q}_{cv} = -1000 \text{ kW} = -1000 \text{ kJ / s}$

$$\Rightarrow \dot{n}_{fuel} = \frac{-10^3 \text{ kJ / s}}{-2112780 \text{ kJ / kmol fuel}} = 4.73 \times 10^{-4} \frac{\text{kmol fuel}}{\text{s}}$$

with $M = 78.11 \frac{\text{kg fuel}}{\text{kmol fuel}}$ from tables;

$$\dot{n}_{fuel} = \left(4.73 \times 10^{-4} \frac{\text{kmol fuel}}{\text{s}} \right) \left(78.11 \frac{\text{kg fuel}}{\text{kmol fuel}} \right) = 0.037 \frac{\text{kg fuel}}{\text{s}}$$

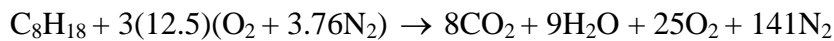
9) A closed, rigid vessel initially contains a gaseous mixture of 1 kmol of Octane (C_8H_{18}) and 300% of theoretical air at $25^\circ C$, 1 atm. If the mixture burns completely, determine the heat transfer from the vessel, in kJ, and the final pressure, in atm, for a final temperature of 1000 K.



Assumptions:

- 1) $W = 0$
- 2) 3.76 moles of N_2 accompany each mole of O_2 in the air and N_2 is inert.
- 3) The ideal gas model is applicable to initial and final mixtures.

Combustion equation for 300% theoretical air is:



The energy balance reduces to $Q - W = U_P - U_R = (U_{\text{final}} - U_{\text{initial}})$

$$\therefore Q = (8\bar{u}_{CO_2} + 9\bar{u}_{H_2O} + 25\bar{u}_{O_2} + 141\bar{u}_{N_2})_f - (1\bar{u}_{C_8H_{18}} + 37.5\bar{u}_{O_2} + 141\bar{u}_{N_2})_i$$

For the ideal gas, $\bar{u} = \bar{h} - \bar{R}T$

Substituting for \bar{u} and collecting $\bar{R}T$ terms,

$$\therefore Q = (8\bar{h}_{CO_2} + 9\bar{h}_{H_2O} + 25\bar{h}_{O_2} + 141\bar{h}_{N_2})_2 - (\bar{h}_{C_8H_{18}} + 37.5\bar{h}_{O_2} + 141\bar{h}_{N_2})_1 - \bar{R}(183T_2 - 179.5T_1)$$

Define $\bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$ and $\bar{h}_f^\circ = 0$ for O_2 and N_2 . Using data from tables:

$$\begin{aligned} \therefore Q = & 8[-395520 + 42769 - 9364]_{CO_2} + 9[-241820 + 35882 - 9904]_{H_2O} \\ & + 25[31389 - 3682]_{O_2} + 141[30129 - 8669]_{N_2} - (-208450)_{C_8H_{18(g)}} \\ & - 8.314[(183)(1000) - (179.5)(298)] \end{aligned}$$

$$\Rightarrow Q = -2098251 \text{ kJ}$$

Using the ideal gas equation for a constant volume process,

$$p_1 V = n_1 \bar{R} T_1 \quad \text{and} \quad p_2 V = n_2 \bar{R} T_2$$

$$\frac{p_2}{p_1} = \left(\frac{n_2}{n_1}\right)\left(\frac{T_2}{T_1}\right) = \left(\frac{183}{179.5}\right)\left(\frac{1000}{298}\right) = 3.421$$

$$p_2 = (3.421)p_1 = 3.421(1) = 3.421 \text{ atm}$$

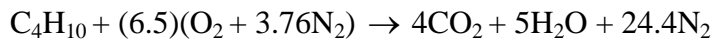
10) Determine the enthalpy of combustion for gaseous butane (C_4H_{10}), in kJ per kmol of fuel and kJ per kg of fuel, at $25^\circ C$, 1 atm, determine:

- a) water vapor in the products.
- b) liquid vapor in the products.

Assumptions:

- 1) Combustion is with the theoretical amount of air.
- 2) 3.76 moles of N_2 accompany each mole of O_2 in the air and N_2 is inert.
- 3) The combustion air and the combustion products can be modeled as ideal gases.

Complete combustion of C_4H_{10} with the theoretical amount of air is described by:



The enthalpy of combustion is:

$$\bar{h}_{RP} = 4(\bar{h}_f^o)_{CO_2} + 5(\bar{h}_f^o)_{H_2O} - (\bar{h}_f^o)_{C_4H_{10}}$$

a) H_2O is a vapor:

$$\bar{h}_{RP} = 4(-393520) + 5(-241820) - (-126150) = -2.657 \times 10^6 \text{ kJ/kmol}(C_4H_{10})$$

For C_4H_{10} , $M = 58.12 \frac{\text{kg fuel}}{\text{kmol fuel}}$, so;

and

$$\bar{h}_{RP} = \frac{-2.657 \times 10^6}{58.12} = -45716 \text{ kJ/kg}$$

a) H_2O is a liquid:

$$\bar{h}_{RP} = 4(-393520) + 5(-285830) - (-126150) = -2.877 \times 10^6 \text{ kJ/kmol}(C_4H_{10})$$

For C_4H_{10} , $M = 58.12 \frac{\text{kg fuel}}{\text{kmol fuel}}$, so;

and

$$\bar{h}_{RP} = \frac{-2.877 \times 10^6}{58.12} = -49501 \text{ kJ/kg}$$

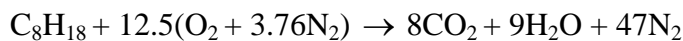
- 11)** Determine the higher heating value, in kJ per kmol of fuel and in kJ per kg of fuel, at 25°C, 1 atm for
- liquid octane (C₈H₁₈).
 - gaseous hydrogen (H₂).
 - liquid methanol (CH₃OH).
 - gaseous butane (C₄H₁₀).

Assumptions:

- Combustion is with the theoretical amount of air.
- 3.76 moles of N₂ accompany each mole of O₂ in the air and N₂ is inert.
- The combustion air and the gaseous products can be modeled as ideal gases. The water formed on combustion is a liquid.

- a) Liquid octane (C₈H₁₈):

The reaction equation for complete combustion with the theoretical amount of air is:



The enthalpy of combustion at 25°C, 1atm is:

$$\bar{h}_{\text{RP}} = 8(\bar{h}_f^\circ)_{\text{CO}_2} + 9(\bar{h}_f^\circ)_{\text{H}_2\text{O(l)}} - (\bar{h}_f^\circ)_{\text{C}_8\text{H}_{18}}$$

$$\bar{h}_{\text{RP}} = 8(-393520) + 9(-285830) - (-249910) = -5470720 \text{ kJ / kmol (C}_8\text{H}_{18}\text{)}.$$

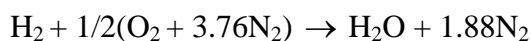
Accordingly;

$$\overline{\text{HHV}} = 5470720 \text{ kJ / kmol (C}_8\text{H}_{18}\text{)}$$

$$\text{HHV} = \frac{5470720 \text{ kJ / kmol (C}_8\text{H}_{18}\text{)}}{114.22 \text{ kg / kmol (C}_8\text{H}_{18}\text{)}} = 47896 \text{ kJ / kg (C}_8\text{H}_{18}\text{)}$$

- b) gaseous hydrogen (H₂):

The reaction equation for complete combustion with the theoretical amount of air is:



The enthalpy of combustion at 25°C, 1atm is:

$$\bar{h}_{\text{RP}} = (\bar{h}_f^\circ)_{\text{H}_2\text{O(l)}} = -285300 \text{ kJ / kmol (H}_2\text{)}$$

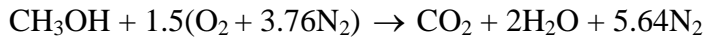
Accordingly;

$$\overline{\text{HHV}} = 285300 \text{ kJ / kmol}(\text{H}_2)$$

$$\text{HHV} = \frac{285300 \text{ kJ / kmol}(\text{H}_2)}{2.018 \text{ kg / kmol}(\text{H}_2)} = 141378 \text{ kJ / kg}(\text{H}_2)$$

c) Liquid methanol (CH_3OH):

The reaction equation for complete combustion with the theoretical amount of air is:



The enthalpy of combustion at 25°C , 1 atm is:

$$\bar{h}_{\text{RP}} = (\bar{h}_f^\circ)_{\text{CO}_2} + 2(\bar{h}_f^\circ)_{\text{H}_2\text{O}(\text{l})} - (\bar{h}_f^\circ)_{\text{CH}_3\text{OH}}$$

$$\bar{h}_{\text{RP}} = 2(-285830) + (-393520) - (-238810) = -726370 \text{ kJ / kmol}(\text{CH}_3\text{OH})$$

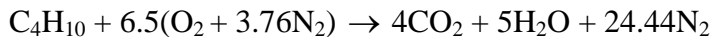
Accordingly;

$$\overline{\text{HHV}} = 726370 \text{ kJ / kmol}(\text{CH}_3\text{OH})$$

$$\text{HHV} = \frac{726370 \text{ kJ / kmol}(\text{CH}_3\text{OH})}{32.05 \text{ kg / kmol}(\text{CH}_3\text{OH})} = 22664 \text{ kJ / kg}(\text{CH}_3\text{OH})$$

d) Gaseous butane (C_4H_{10}):

The reaction equation is:



The enthalpy of combustion at 25°C , 1 atm is:

$$\bar{h}_{\text{RP}} = 4(\bar{h}_f^\circ)_{\text{CO}_2} + 5(\bar{h}_f^\circ)_{\text{H}_2\text{O}(\text{l})} - (\bar{h}_f^\circ)_{\text{C}_4\text{H}_{10}(\text{g})}$$

$$\bar{h}_{\text{RP}} = 4(-393520) + 5(-285830) - (-126150) = -2877080 \text{ kJ / kmol}(\text{C}_4\text{H}_{10})$$

Accordingly;

$$\overline{\text{HHV}} = 2877080 \text{ kJ / kmol}(\text{C}_4\text{H}_{10})$$

$$\text{HHV} = \frac{2877080 \text{ kJ / kmol}(\text{C}_4\text{H}_{10})}{58.12 \text{ kg / kmol}(\text{C}_4\text{H}_{10})} = 49502 \text{ kJ / kg}(\text{C}_4\text{H}_{10})$$

12) For each of the following fuels, determine the adiabatic flame temperature, in K, for complete combustion with 200% of theoretical air in a combustor operating at steady state. The reactants enter at 25°C, 1 atm.

- carbon
- hydrogen (H₂).
- liquid octane (C₈H₁₈).

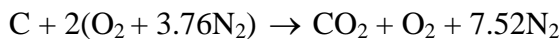


Assumptions:

- For the control volume shown in the accompanying figure $\dot{Q}_{cv} = \dot{W}_{cv} = 0$ and the effects of kinetic and potential energy are negligible.
- 3.76 moles of N₂ accompany each mole of O₂ in the air and N₂ is inert.
- The ideal gas model is applicable to the combustion air and products.

a) Fuel is carbon (C):

Complete combustion of C with the 200% of theoretical amount of air is described by:



An energy rate balance at steady state reduces to:

$$0 = \frac{\dot{Q}_{cv}}{\dot{n}_{fuel}} - \frac{\dot{W}_{cv}}{\dot{n}_{fuel}} + (\bar{h}_C)_1 + (2\bar{h}_{O_2} + 7.52\bar{h}_{N_2})_2 - (\bar{h}_{CO_2} + \bar{h}_{O_2} + 7.52\bar{h}_{N_2})_3$$

with $\bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$ and noting that $\bar{h}_f^\circ = 0$ for C, O₂ and N₂.

$$\Rightarrow 0 = -[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{CO_2} - [\bar{h}(T_3) - \bar{h}(298)]_{O_2} - 7.52[\bar{h}(T_3) - \bar{h}(298)]_{N_2}$$

Accordingly, with data from the ideal gas tables:

$$\bar{h}_{CO_2}(T_3) + \bar{h}_{O_2}(T_3) + 7.52\bar{h}_{N_2}(T_3) = -(-393520 - 9364) + 8682 + 7.52(8669) = 476757$$

Solving, $T_3 \cong 1506$ K

b) Fuel is hydrogen (H₂):

Complete combustion of H₂ with the 200% of theoretical amount of air is described by:



An energy rate balance at steady state reduces to:

$$0 = \frac{\dot{Q}_{\text{cv}}}{\dot{n}_{\text{fuel}}} - \frac{\dot{W}_{\text{cv}}}{\dot{n}_{\text{fuel}}} + (\bar{h}_{\text{H}_2})_1 + (\bar{h}_{\text{O}_2} + 3.76\bar{h}_{\text{N}_2})_2 - (\bar{h}_{\text{H}_2\text{O}} + 1/2\bar{h}_{\text{O}_2} + 3.76\bar{h}_{\text{N}_2})_3$$

with $\bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$ and noting that $\bar{h}_f^\circ = 0$ for H_2 , O_2 and N_2 .

$$\Rightarrow 0 = -[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{H}_2\text{O}} - \frac{1}{2}[\bar{h}(T_3) - \bar{h}(298)]_{\text{O}_2} - 3.76[\bar{h}(T_3) - \bar{h}(298)]_{\text{N}_2}$$

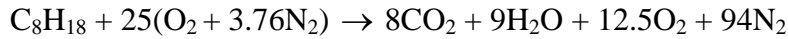
Accordingly, with data from the ideal gas tables:

$$\bar{h}_{\text{H}_2\text{O}}(T_3) + \frac{1}{2}\bar{h}_{\text{O}_2}(T_3) + 3.76\bar{h}_{\text{N}_2}(T_3) = -(-241820 - 9904) + \frac{1}{2}(8682) + 3.76(8669) = 288660$$

Solving, $T_3 \cong 1647 \text{ K}$

c) Fuel is liquid octane (C_8H_{18}):

Complete combustion of C_8H_{18} with the 200% of theoretical amount of air is described by:



An energy rate balance at steady state reduces to:

$$0 = \frac{\dot{Q}_{\text{cv}}}{\dot{n}_{\text{fuel}}} - \frac{\dot{W}_{\text{cv}}}{\dot{n}_{\text{fuel}}} + (\bar{h}_{\text{C}_8\text{H}_{18}})_1 + (25\bar{h}_{\text{O}_2} + 94\bar{h}_{\text{N}_2})_2 - (8\bar{h}_{\text{CO}_2} + 9\bar{h}_{\text{H}_2\text{O}} + 12.5\bar{h}_{\text{O}_2} + 94\bar{h}_{\text{N}_2})_3$$

with $\bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$ and noting that $\bar{h}_f^\circ = 0$ for O_2 and N_2 .

$$\Rightarrow 0 = [\bar{h}_f^\circ]_{\text{C}_8\text{H}_{18}} - 8[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{CO}_2} - 9[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{H}_2\text{O}} - 12.5[\bar{h}(T_3) - \bar{h}(298)]_{\text{O}_2} - 94[\bar{h}(T_3) - \bar{h}(298)]_{\text{N}_2}$$

Accordingly, with data from the ideal gas tables:

$$\begin{aligned} 8\bar{h}_{\text{CO}_2}(T_3) + 9\bar{h}_{\text{H}_2\text{O}}(T_3) + 12.5\bar{h}_{\text{O}_2}(T_3) + 94\bar{h}_{\text{N}_2}(T_3) &= -249910 - 8(-393520 - 9364) \\ &\quad - 9(-241820 - 9904) + 12.5(8682) + 94(8669) \\ &= 6162089 \end{aligned}$$

Solving, $T_3 \cong 1507 \text{ K}$

13) Methane (CH₄) at 25°C 1 atm enters an insulated reactor operating at steady state and burns with the theoretical amount of air entering at 25°C, 1 atm. Determine the temperature of the exiting combustion products if:

a) combustion is complete.

b) 90% of the carbon in the fuel burns to CO₂ and the rest burns to CO.

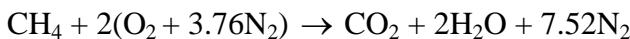
Neglect kinetic and potential energy effects.



Assumptions:

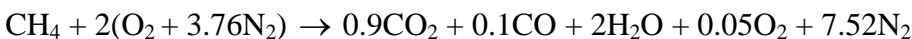
- 1) For the control volume shown in the accompanying figure operates at steady state with $\dot{Q}_{cv} = \dot{W}_{cv} = 0$ and the effects of kinetic and potential energy is negligible.
- 2) 3.76 moles of N₂ accompany each mole of O₂ in the air and N₂ is inert.
- 3) The ideal gas model is applicable to the combustion air and products.
- 4) Combustion is with the theoretical amount of air.

a) Complete combustion of CH₄ with the theoretical amount of air is described by:



An energy balance gives: $T_2 \cong 2328 \text{ K}$

b) Combustion of CH₄ with 90% of the carbon in the fuel going to CO₂ is described by:



An energy rate balance at steady state reduces to give:

$$0 = \frac{\dot{Q}_{cv}}{\dot{n}_{\text{fuel}}} - \frac{\dot{W}_{cv}}{\dot{n}_{\text{fuel}}} + (\bar{h}_{\text{CH}_4})_1 + (2\bar{h}_{\text{O}_2} + 7.52\bar{h}_{\text{N}_2})_2 - (0.9\bar{h}_{\text{CO}_2} + 0.1\bar{h}_{\text{CO}} + 2\bar{h}_{\text{H}_2\text{O}} + 0.05\bar{h}_{\text{O}_2} + 7.52\bar{h}_{\text{N}_2})_3$$

with $\bar{h} = \bar{h}_f^\circ + \Delta\bar{h}$ and noting that $\bar{h}_f^\circ = 0$ for O₂ and N₂. This becomes:

$$\begin{aligned} \Rightarrow 0 = & [\bar{h}_f^\circ]_{\text{CH}_4} - 0.9[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{CO}_2} - 0.1[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{CO}} \\ & - 2[\bar{h}_f^\circ + \bar{h}(T_3) - \bar{h}(298)]_{\text{H}_2\text{O}} - 0.05[\bar{h}(T_3) - \bar{h}(298)]_{\text{O}_2} - 7.52[\bar{h}(T_3) - \bar{h}(298)]_{\text{N}_2} \end{aligned}$$

Solving,

$$0.9\bar{h}_{\text{CO}_2}(T_3) + 0.1\bar{h}_{\text{CO}}(T_3) + 2\bar{h}_{\text{H}_2\text{O}}(T_3) + 0.05\bar{h}_{\text{O}_2}(T_3) + 7.52\bar{h}_{\text{N}_2}(T_3) = [\bar{h}_f^\circ]_{\text{CH}_4} - 0.9[\bar{h}_f^\circ - \bar{h}(298)]_{\text{CO}_2} \\ - 0.1[\bar{h}_f^\circ - \bar{h}(298)]_{\text{CO}} - 2[\bar{h}_f^\circ - \bar{h}(298)]_{\text{H}_2\text{O}} - 0.05\bar{h}_{\text{O}_2}(298) - 7.52\bar{h}_{\text{N}_2}(298)$$

With data from the ideal gas tables:

$$0.9\bar{h}_{\text{CO}_2}(T_3) + 0.1\bar{h}_{\text{CO}}(T_3) + 2\bar{h}_{\text{H}_2\text{O}}(T_3) + 0.05\bar{h}_{\text{O}_2}(T_3) + 7.52\bar{h}_{\text{N}_2}(T_3) = -74850 - 0.9(-393520 - 9364) \\ - 0.1(-110530 - 8669) - 2(-241820 - 9904) + 0.05(8682) + 7.52(8669) \\ = 868739$$

Solving, $T_3 \cong 2265 \text{ K}$